



Transport and Climate Change: Reasons, Resources, Responses

While federal politicians argue over who has the best climate change policy and scientists continue to refine the climate change science and warnings, transport professionals are left wondering what it means for them, and how they should respond to the climate change challenge. This forum explores the reasons for climate change, some climate-friendly technologies already available, and policy responses relevant to the transport sector.



OPTIMISATION OF DUAL FUEL (ETHANOL-DIESEL) ENGINE



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Presentation Outline

- Background
- Introduction to Engines
- Introduction to Dual Fuel Systems
- Proposed Dual Fuel System
- Research Approaches
- Conclusions

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Background: Liquid Fuel Alternatives

- What can serve as an alternate fuel?
 - Alcohol (Ethanol, Methanol)
 - Vegetable oil
 - Liquefied Petroleum Gas (LPG)
 - Compressed Natural Gas (CNG)
 - Liquefied Natural Gas (LNG)

Fuel Prices in Brisbane

| | Calorific Value (MJ/kg) | Nov 2006 | Dec 2006 | Jan 2007 | Feb 2007 | Mar 2007 | Apr 2007 | May 2007 | Jun 2007 |
|----------|-------------------------|----------|----------|----------|----------|----------|----------|----------|----------|
| Unleaded | 47.3 | 108.1 | 110.9 | 109.1 | 107.8 | 117.5 | 119.5 | 123.8 | 122.5 |
| Premium | 47.3 | 116.3 | 103.7 | 116.5 | 114.4 | 126.2 | 127.0 | 131.1 | 130.1 |
| Diesel | 44.8 | 116.7 | 112.7 | 116.1 | 114.5 | 116.1 | 119.7 | 121.2 | 122.4 |
| LPG | 49.9 | 52.6 | 41.9 | 54.5 | 53.7 | 53.2 | 54.4 | 54.7 | 54.3 |
| Ethanol | 29.7 | | | 107.9 | 106.2 | 116.3 | 119.0 | 122.9 | 121.3 |

http://www.racq.com.au/cps/rde/xchg/racq/cms_production/hs.xsl/News_Fuel_Prices_Foun_news_fuel_pricereports_ENA_HTML.htm

Dual Fuel Systems

- Fumigation
 - Addition of secondary fuel to the intake air charge
 - Displacing up to 50% of the diesel fuel demand
- Dual injection
 - Separate injection system with control for each fuel
 - Displacing up to 90% of diesel engine demand
- Blend Fuel
 - Mixture of the fuel just prior to injection
 - Displacing up to 25% of diesel engine demand
- Fuel Emulsion
 - Mixture of the fuel by using emulsifier
 - Displacing up to 25% of diesel engine demand

Quoted from Abu-Audais, et al., *Energy Conversion & Management* 2000

Ethanol

CCO

- Ethyl alcohol (C₂H₅OH)
- Production of Ethanol
 - Carbohydrate-rich plants
 - Ligno-cellulosic products

World Ethanol Production, 1980–2004
Source: IEA

| Raw material | Carbohydrate | | Ethanol | |
|---------------------|--------------|----|---------|---------|
| | (t/ha) | % | (l/t) | (hl/ha) |
| Beet | 40–50 | 16 | 90–100 | 38–48 |
| Sugar cane | 50–100 | 13 | 60–80 | 35–70 |
| Maize | 4–8 | 60 | 360–400 | 15–30 |
| Wheat | 25 | 62 | 370–420 | 8–20 |
| Barley | 2–4 | 52 | 310–350 | 7–13 |
| Grain sorghum | 2–5 | 70 | 330–370 | 7–18 |
| Potatoes | 20–30 | 18 | 100–120 | 22–33 |
| Sweet potato | 10–20 | 26 | 140–170 | 16–31 |
| Cassava | 12–15 | 27 | 175–190 | 22–23 |
| Jerusalem artichoke | 30–60 | 17 | 80–100 | 27–54 |

| Raw material (hydrolytic agent) | Dry matter (t/ha) | Ethanol | |
|------------------------------------|----------------------|---------|---------|
| | | (l/t) | (hl/ha) |
| Softwood | (dilute acids) | 9–15 | 180–220 |
| | (concentrated acids) | 9–15 | 230–270 |
| Hardwood | (dilute acids) | 9–15 | 160–180 |
| | (concentrated acids) | 9–15 | 190–220 |
| Straw | (dilute acids) | 1.5–3.5 | 140–160 |
| | (concentrated acids) | 1.5–3.5 | 160–180 |

<http://www.rise.org.au/info/Applic/Ethanol/index.html>

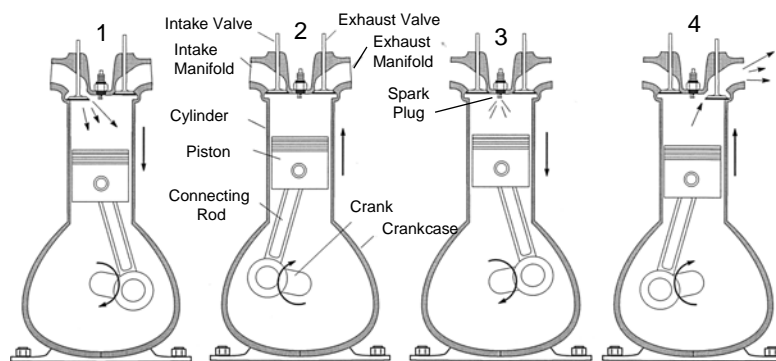
Ethanol Fuel

- Application of Ethanol
 - Gasoline blend (5 – 10%)
 - Diesel blend (E-diesel)
- Characteristic of Ethanol
 - Lower volumetric energy content
 - High octane rating
 - result in increased engine efficiency and performance
- Environmental characteristics
 - Produced “renewably”
 - Reducing greenhouse gas emission

Introduction to Engine

- External combustion
 - combustion of an air-fuel mixture transfers heat to a second fluid which becomes the motive (working) fluid that produces power
 - E.g., steam driven engine
- Internal combustion
 - the products of combustion are the motive fluid
 - Spark ignition (SI) engines
 - a compressed, homogeneous air-fuel mixture (15:1 ratio of air to fuel by mass) is ignited using a spark
 - Petroleum (high octane rating) is used
 - Compression ignition (CI) engines
 - rapid compression of air to a high pressure raises the temperature so that fuel, when delivered into combustion chamber, spontaneously ignites without need for a spark
 - often referred to as a Diesel engine
 - Diesel fuel (high cetane rating) is used

Four Stroke Engine



Intake Stroke
Intake valve opens, admitting fuel and air. Exhaust valve closed for most of stroke

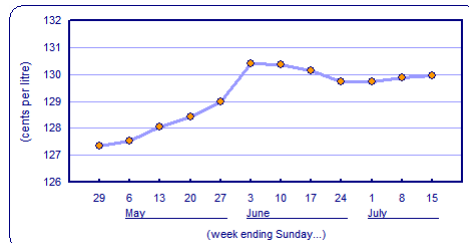
Compression Stroke
Both valves closed, Fuel/air mixture is compressed by rising piston. Spark ignites mixture near end of stroke.

Power Stroke
Fuel-air mixture burns, increasing temperature and pressure, expansion of combustion gases drives piston down. Both valves closed - exhaust valve opens near end of stroke

Exhaust Stroke
Exhaust valve open, exhaust products are displaced from cylinder. Intake valve opens near end of stroke.

Introduction to Dual Fuel Systems

- Why use dual fuel?
 - Transition technology
 - Fossil (non-renewable) resource
 - Diesel price
 - Particle and No_x emission
 - Higher combustion temperature than Petroleum
 - Greenhouse gas emission

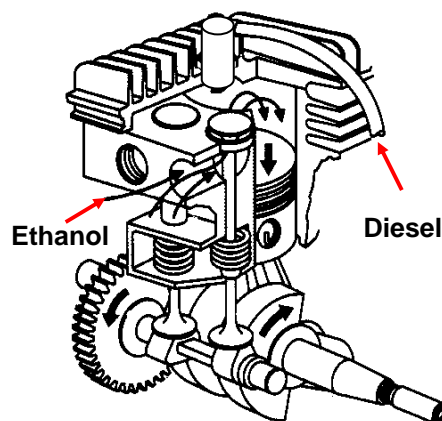


Average Weekly Diesel Fuel Prices for the 12 Weeks to Sunday, 15 July 2007
- National Average -
<http://www.aip.com.au/pricing/retail/diesel/index.htm>

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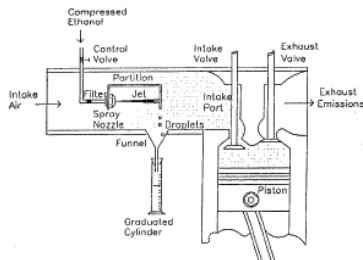
Introduction to Ethanol fumigation System

- Pilot Fuel
 - Injected from spray nozzle
 - Reliable and instant ignition
 - High cetane, diesel fuel
- Supplemental Fuel
 - Injected through air-intake
 - Long and slow combustion
 - High octane, ethanol
- Advantage
 - Minimum modification needed
 - Ethanol fuel is separate from the diesel system, no mixing needed
 - Ethanol could reduce smoke
 - Fumigation can substitute ethanol for 50% diesel fuel

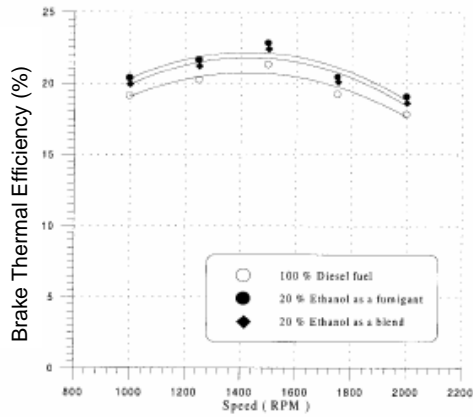


Comparison between Fumigation and Blend

- Comparison Test (Abu-Qudais et al. 2000)
 - Single cylinder, four stroke, direct injection diesel engine
 - Swept volume 0.582 litre

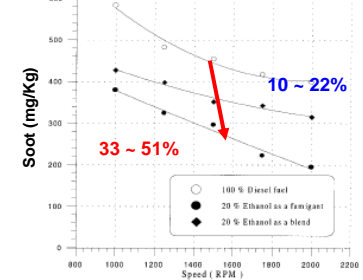
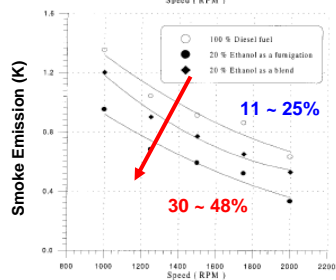
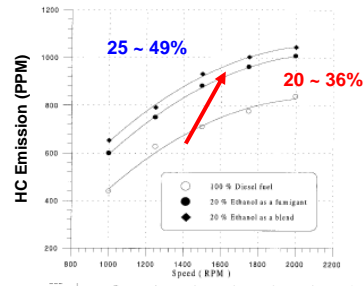
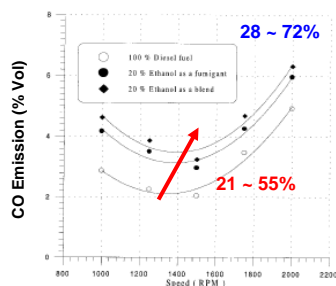


Ethanol Fumigation System



- Slight improvement in the efficiency
 - 7.5% for 20% ethanol fumigation
 - 5.4% for 20% ethanol-diesel fuel blends

Emission Comparison between Fumigation and Blend

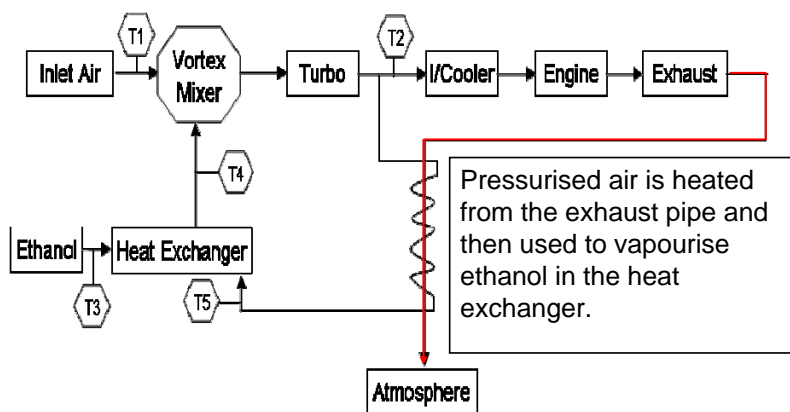


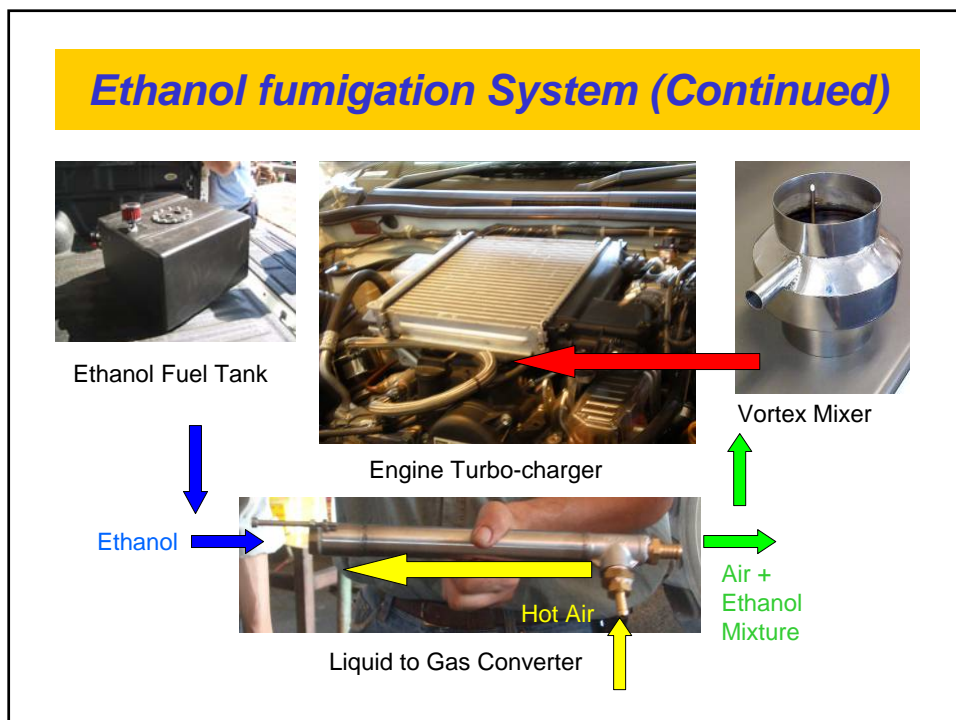
Dual Fuel System Research Team

Mr Uli Kruger, the inventor



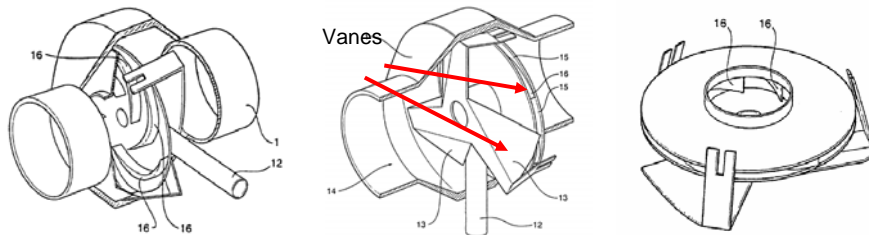
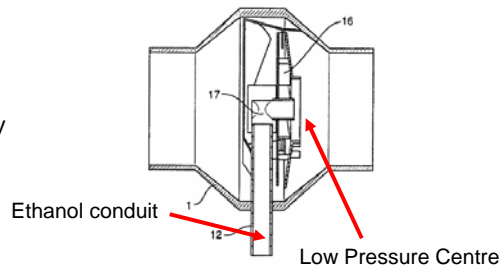
Schematic Diagram of the Ethanol fumigation System





Natural Vortex Mixer

- Also called Centripetal Vortex, inward directed spiral
- Feature: low pressure at centre
- Can provide significant reliability in respect of air-fuel ratio over a reasonable load range



Dual Fuel System Test Run

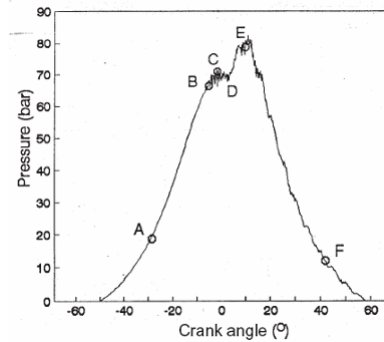
- Test Parameters
 - Engine: Perkins T4-236
 - 52 kW generator
 - 3000 rpm
 - Displacement: 3.86 liters
 - Compression ratio: 16:1
 - Supplemental fuel: LPG
- Test Result
 - Relative consistency in the diesel power ratio over total power
 - LPG mass percentage keep constant over the range of power outputs.
 - Significant reduction in No and smoke
 - Marginal drop in overall thermal efficiency

| LOAD kW | time/200 mls | kW of Diesel | mass of LPG kg | kW of LPG | Efficiency % | Diesel Mass reduc % | Energy ratio Diesel/Total |
|---------|--------------|--------------|----------------|-----------|--------------|---------------------|---------------------------|
| 20 | 109 | 64.7 | 0 | 0 | 30.9 | 100 | 1.00 |
| 20 | 214 | 33.0 | 0.284 | 61.4 | 21.2 | 49.1 | 0.35 |
| 30 | 77 | 91.6 | 0 | 0 | 32.7 | 100 | 1.00 |
| 30 | 168 | 42.0 | 0.254 | 70.0 | 26.8 | 54.2 | 0.37 |
| 40 | 59 | 119.6 | 0 | 0 | 33.5 | 100 | 1.00 |
| 40 | 125 | 56.4 | 0.192 | 71.1 | 31.4 | 52.8 | 0.44 |

| LOAD kW | CO2 % | CO2 reduc % | NO ppm | NO reduc % | Opacity % | Opacity red % | AFR | HC |
|---------|-------|-------------|--------|------------|-----------|---------------|------|-----|
| 20 | 6.23 | | 366 | | 16 | | 34.2 | 0 |
| 20 | 5.8 | 6.9 | 221 | 39.6 | 4 | 75.0 | 30.5 | |
| 30 | 8.18 | | 645 | | 34 | | 26.6 | 0 |
| 30 | 7.47 | 8.7 | 400 | 38.0 | 14 | 58.8 | 26.3 | |
| 40 | 8.71 | | 938 | | 50 | | 22.7 | 0 |
| 40 | 7.88 | 9.5 | 553 | 41.0 | 33 | 34.0 | 23.3 | 202 |

Research Approaches

- 1. Developing a framework for dual fuel engine evaluation
 - Multivariate analysis
 - Optimum engine operating parameter
 - Dual fuel/water ratio
 - Objectives
 - Reduce environmental emissions
 - Maximize power output and performance
- 2. Thermodynamic modeling
 - Modeling of dual fuel fumigation system
 - Modeling of diesel engine
 - Premixed ethanol fuel
 - Addition of water to the substitution fuel

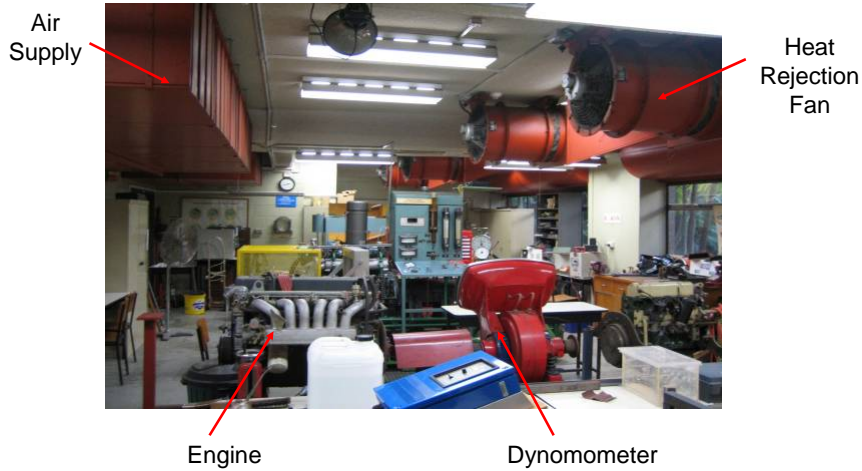


Dual fuel pilot injection pressure-crank angle diagram
AB: pilot ignition delay (AB)
BC: pilot premixed combustion (BC)
CD: primary fuel delay period (CD)
DE: rapid combustion of primary fuel (DE)
EF: diffusion combustion stage (EF)
(After Nwafor, 2002)

Research Approaches (Continued)

- 3. Laboratory optimization, performance and emission testing
 - Engine performance testing
 - Throttle setting of 20, 40, 60, 80, and 100%
 - Dual fuel ratios of "low", "medium", and "high"
 - Water ratio of "low", "medium", and "high"
 - 3 repetitions for statistical stability
 - Laboratory fuel/water optimization and performance testing
 - Engine performance over the full range of operating parameter
 - Primary measurement: brake power, peak and mean effective pressure
 - Emission testing
 - Gaseous emission: CO, CO₂, NO_x, and unburned hydrocarbons (HCs)
 - Particulate emissions, using mini dilution system and Scanning Mobility Particle Sizer (SMPS)
 - Standard industry 200 hour test
- 4. Identifying optimal engine/fuel parameter
 - Achieve optimal engine performance without significant increase of emissions
 - Application of after-treatment technologies possible
 - Elimination of knock (pre-ignition)
- 5. Field testing
 - Dalby City Council
 - Rocky point sugar Mill
- 6. Transfer of knowledge

Engine Performance Testing



Timeline

| Table 1: Project Schedule | | 2007 | 2008 | 2009 |
|---------------------------|--|--------|--------|--------|
| | | Year 1 | Year 2 | Year 3 |
| Module 1 | 1.1 Framework for dual fuel engine evaluation | | | |
| | 1.2 multicriteria decision making | | | |
| | 1.3 elimination of redundant parameters | | | |
| Module 2 | Thermodynamic modelling | | | |
| | 2.1 Consolidation of Teakle (2004) model | | | |
| | 2.2 Comparison with other models eg Dibble et al , 2001 | | | |
| | 2.3 Investigation of engine characteristics, knock, efficiency | | | |
| | 2.4 Modelling of emissions: gaseous and particulate | | | |
| Module 3 | Laboratory performance and emission testing | | | |
| | 3.1 Engine performance testing | | | |
| | 3.2 Laboratory fuel/water optimisation and performance testing | | | |
| | 3.3 Emission testing | | | |
| | 3.4 Standard industry 200 hour test | | | |
| Mod 4 | 4. Identifying optimal engine/fuel parameters | | | |
| Mod 5 | 5 Field testing | | | |
| Mod 6 | 6 Transfer of knowledge | | | |

Conclusions

- Dual fuel systems were introduced
- Ethanol fumigation method was proposed
- Preliminary test show that this method with a natural vortex mixer can achieve consistent alternate fuel ratio over large load ranges
- Extensive rigorous research will be conducted to reach optimum performance